

# **Universität Stuttgart** Fakultät für Bau- und

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## **Motivation**

Polygonal elements have shown to be advantageous in many aspects of the finite element technology in structural engineering. They provide:

- High flexibility in meshing arbitrary geometries.
- Local mesh refinement is easily applicable.
- No restriction on the number of edges per element.

The Mixed Displacement method and Polygonal Plate **Finite Elements** 

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Reissner-Mindlin plates, purely displacement-based For formulations are prone to shear locking phenomena. This issue is treated with the mixed displacement method, which is:

- A formulation that returns intrinsically locking free results by equal order interpolation.
- Locking free, independent of the discretization scheme.

### **Scaled Boundary Finite Elements**

A scaled boundary element is partitioned into sectors by connecting all boundary vertices with the scaling center O. The local coordinate system is defined by the scaling coordinate  $\xi$ and the circumferential coordinate  $\eta$ , where

 $\eta \in [-1,1]$  and  $\xi \in [0,1]$ .



# **Displacements of the Simply Supported Plate Problem**

Polygonal meshes can be generated through centroidal Voronoi tessellation (CVT). To create well-shaped, uniformly sized elements, the Lloyd algorithm is an essential component of the CVT procedure, significantly impacting the solution.



center of the plate against the number

of Lloyd iterations with a slenderness

Fig. 1: Scaled boundary element with local coordinate system partitioned into sectors  $V^e$ 

## **Mixed Displacement Method**

The mixed displacement principle can be expressed as a multifield functional in terms of the displacement field  $\mathbf{u}$  and an additional field  $\bar{\mathbf{u}}$  that represents the integral of the shear strains. These additional terms remove geometrical locking effects intrinsically. The internal part of the functional which is derived from the Hellinger-Reissner functional reads:

$$\Pi_{\rm HR}^{\rm int}(\mathbf{u},\bar{\mathbf{u}}) = \int \left(\frac{1}{2} \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}}^T \mathbf{C} \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}} - \boldsymbol{\varepsilon}_{\mathbf{u}}^T \mathbf{C} \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}}\right) d\Omega$$

displacement errors of the plate domain  $e_{L^2}^w$  against the number of elements with a slenderness of 100

### Results

• The mixed displacement method with scaled boundary parametrization returns far better results for both structured and unstructured Voronoi meshes compared to the standard, purely displacement based finite element formulation.

of 100

- The uniformity and smoothness of the polygonal mesh has a significant impact on the results.
- Observed shear force oscillations suggest that the current implementation has not resolved all locking effects entirely.
- It is generally possible to place the scaling center on arbitrary locations inside and outside the respective element for a pure scaled boundary finite element formulation.

### Literature

The constraints on unstructured polygonal meshes can be enforced via a penalty method. The first variation of the complete functional reads:

$$\delta\Pi_{\mathrm{HR}}^{\mathrm{int}}(\mathbf{u},\bar{\mathbf{u}}) = \int_{\Omega} \left( \delta \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}}^T \mathbf{C} \, \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}} - \delta \boldsymbol{\varepsilon}_{\mathbf{u}}^T \mathbf{C} \, \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}} - \boldsymbol{\varepsilon}_{\mathbf{u}}^T \mathbf{C} \, \delta \boldsymbol{\varepsilon}_{\bar{\mathbf{u}}} - \epsilon_p^T \mathbf{C} \, \boldsymbol{\varepsilon}_p^T \mathbf{C} \, \boldsymbol{\varepsilon}_p \right) \mathrm{d}\Omega$$

Supervision

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